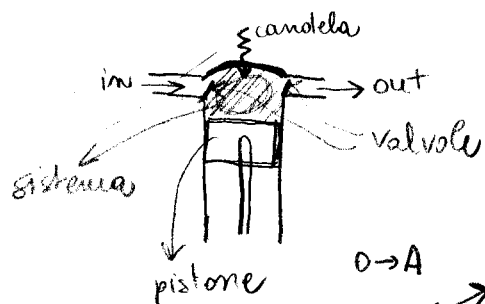


# Esercizio sul motore a scoppio



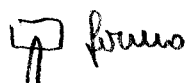
Ciclo del motore a scoppio (o a benzina)



$O \rightarrow A$  0) entra da "in" aria + benzina  
 premessa Tengono a pressione cost. e il pistone scende.



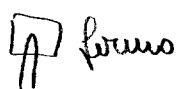
$A \rightarrow B$  1) il pistone comprime adiabaticamente



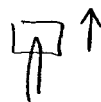
$B \rightarrow C$  2) scintilla dalla candela  $\Rightarrow$  da  $Q$  a volume costante



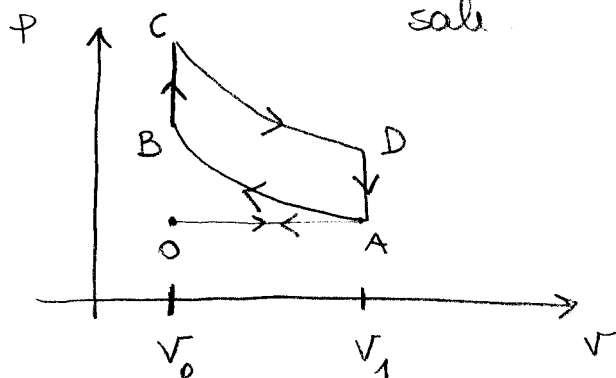
$C \rightarrow D$  3) pistone scende: espansione adiabatica



$D \rightarrow A$  4) si apre la valvola out. Energia espulsa molto rapidamente



$A \rightarrow O$  5) miscela espulsa col pistone che sale.

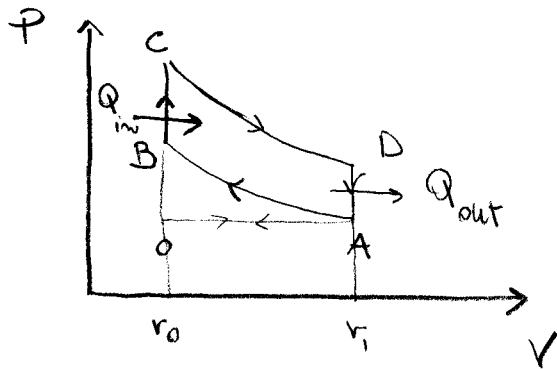


Piano P-V  
 Ciclo di Otto

Motori a 4 tempi: { aspirazione  
 compressione  
 scoppio  
 scarico }  $\rightarrow$  lavoro uguale e opposto

Calcoliamo  $\eta$ :

2



$$\eta = \frac{\Delta L}{Q_{in}}$$

$$0 = \Delta U = \Delta Q - \Delta L \Rightarrow \Delta L = \Delta Q = Q_{Be} + Q_{DA}$$

$$Q_{Be} = n C_v (T_c - T_B)$$

$$Q_{DA} = n C_v (T_A - T_D) \Rightarrow \Delta L = n C_v (T_c - T_B + T_A - T_D)$$

$$\eta = \frac{\Delta L}{Q_{Be}} = 1 + \frac{T_A - T_D}{T_c - T_B} = 1 - \frac{T_D - T_A}{T_c - T_B}$$

Vogliamo ora esprimere  $\eta$  in funzione di  $V_A$  e  $V_B$

$$A \rightarrow B \quad T_A V_A^{\gamma-1} = T_B V_B^{\gamma-1}$$

$$C \rightarrow D \quad T_C V_C^{\gamma-1} = T_D V_D^{\gamma-1}$$

$$\begin{cases} T_A V_1^{\gamma-1} = T_B V_0^{\gamma-1} \\ T_C V_0^{\gamma-1} = T_D V_1^{\gamma-1} \end{cases}$$

$$T_A = T_B \left( \frac{V_0}{V_1} \right)^{\gamma-1}$$

$$T_D = T_C \left( \frac{V_2}{V_1} \right)^{\gamma-1}$$

$$\eta = 1 - \frac{(T_c - T_B)}{T_c - T_B} (V_0/V_1)^{\gamma-1} = 1 - \left( \frac{V_0}{V_1} \right)^{\gamma-1}$$

oppure si può scrivere

rendimento del ciclo di

Due caratteristiche di un motore sono importanti:

cilindrata volume coperto dal pistone nel cilindro

rapporto di compressione  $r = \frac{V_A}{V_B} = \frac{V_1}{V_2}$

Facciamo qualche conto: calcoliamo la potenza fornita da un motore a benzina con 6 cilindri, di 3 l che fa 4000 giri/minuto ed ha

$r = 9.5$   $T_A = 27^\circ \text{C}$  (temp ambiente)

$T_C = 1350^\circ \text{C}$  (temp di combustione)

$P_A = 10^5 \text{ Pa}$  (press. atm.)

calcoliamo  $V_A$  e  $V_B$

$$\frac{V_A}{V_B} = r = 9.5$$

$$V_A - V_B = \frac{\text{cil}}{6} = \frac{3 \times 10^{-3} \text{ m}^3}{6} = 0.5 \times 10^{-3} \text{ m}^3 = \Delta V$$

$$\Rightarrow \begin{cases} V_A = r V_B \\ V_B (r - 1) = \Delta V \end{cases} \Rightarrow \begin{cases} V_A = \frac{r}{r-1} \Delta V = 0.559 \times 10^{-3} \text{ m}^3 \\ V_B = \frac{\Delta V}{r-1} = 0.588 \times 10^{-4} \text{ m}^3 \end{cases}$$

Applichiamo  $PV = nRT$

$$\Rightarrow n = \frac{P_A V_A}{R T_A} = 0.02242$$

Se il gas è aria con  $p_m \approx 29$  (20%  $\text{O}_2$  + 80%  $\text{N}_2$ )

$$m = 0.64 \text{ g}$$

A → B adiabatica

$$P_B V_B^\gamma = P_A V_A^\gamma \Rightarrow P_B = \left( \frac{V_A}{V_B} \right)^\gamma P_A = P_A r^\gamma$$

$$\gamma = \frac{7}{5} \Rightarrow P_B = 2.34 \times 10^6 \text{ Pa}$$

$$T_B = \frac{P_B V_B}{nR} \approx 739 \text{ K}$$

B → C isocora ⇒

$$V_C = V_B$$

$$T_C = (1350 + 273) \text{ K} = 1623 \text{ K}$$

$$\Rightarrow P_C = \frac{nRT_C}{V_C} = 5.14 \times 10^6 \text{ Pa}$$

C → D adiabatica

$$V_D = V_A$$

$$P_D = P_C \left( \frac{V_C}{V_A} \right)^\gamma = P_C r^{-\gamma} = 2.20 \times 10^5 \text{ Pa}$$

$$T_D = \frac{P_D V_A}{nR} = 660 \text{ K}$$

$$\eta = \frac{\Delta L}{Q_{Be}} = \dots = 1 - \frac{T_D}{T_C} = 0.59 \Rightarrow \eta = 59\%$$

$$P_{\text{tot}} = 6 \text{ n° cilindri} \times \left[ \frac{4000}{60} \right] \times \Delta L \times \frac{1}{2}$$

n° giri/sec

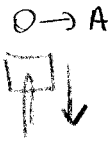
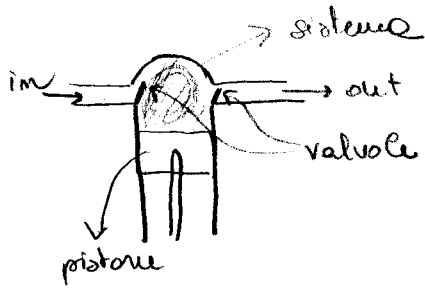
potenza fornita solo  
ogni 2 rivoluzioni

$$\Delta L = Q_{Be} + Q_{DA}$$

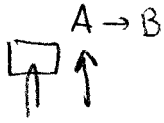
$$= nC_V (T_C - T_B + T_A - T_D) = 244 \text{ J}$$

# Esercizio sul motore diesel

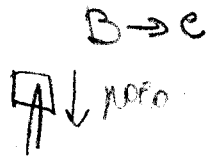
1



0) entro da "in" aria  
P cost. e il pistone scende



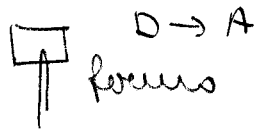
1) compressione adiabatica



2) in B inietto il carburante  
=> aria + carburante si espande  
a pressione costante



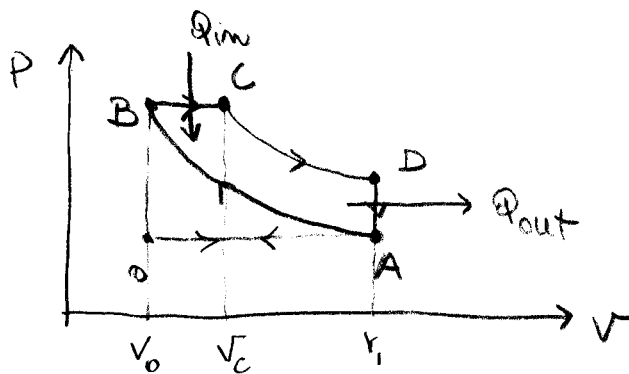
3) alta temp. delle miscele  
causa combustione => scoppio  
e espansione adiabatica



4) "out" aperta e moglie espulsa  
molto rapidamente



5) miscele espulsa con  
pressione che sale



$$\Delta U = 0 \Rightarrow \Delta L = \Delta Q = Q_{BC} + Q_{DA}$$

$$Q_{BC} = m C_p (T_C - T_B) \Rightarrow$$

$$Q_{DA} = m C_p (T_D - T_A)$$

$$\eta = \frac{AL}{Q_{Be}} = \frac{Q_{Be} + Q_{DA}}{Q_{Be}} = 1 + \frac{c_p}{c_p} \frac{T_A - T_D}{T_c - T_B}$$

$$\eta = 1 - \frac{1}{\gamma} \frac{T_D - T_A}{T_c - T_B}$$

$$A \rightarrow B \quad T_A V_A^{\gamma-1} = T_B V_B^{\gamma-1} \quad T_A = T_B \left( \frac{V_B}{V_A} \right)^{\gamma-1}$$

$$c \rightarrow D \quad T_c V_c^{\gamma-1} = T_D V_D^{\gamma-1} \quad T_D = T_c \left( \frac{V_c}{V_D} \right)^{\gamma-1}$$

$$\eta = 1 - \frac{1}{\gamma} \frac{T_c \left( \frac{V_c}{V_1} \right)^{\gamma-1} - T_B \left( \frac{V_2}{V_1} \right)^{\gamma-1}}{T_c - T_B}.$$

Facciamo qualche conto: trovare la potenza fornita da un motore diesel con 4 cilindri, 2 L di cilindrata, 3000 giri al minuto,  $r = \frac{V_A}{V_B} = 22$

$$e \quad \frac{V_c}{V_B} = x = 2 \quad T_A = 300 \text{ K}$$

$$P_A = 10^5 \text{ Pa}$$

$V_A$  e  $V_B$  stesse formule del pb. del motore a scoppio  $\Rightarrow$

$$\begin{cases} V_A = \frac{r}{r-1} \Delta V = 0.523 \cdot 10^{-3} \text{ m}^3 \\ V_B = \frac{\Delta V}{r-1} = 0.238 \cdot 10^{-4} \text{ m}^3 \end{cases}$$

$$m = \frac{P_A V_A}{RT_A} = 0.02101 \approx 0.609 \text{ g di aria}$$

A → B adiabatic

$$P_B = P_A r^\gamma = 7.57 \cdot 10^6 \text{ Pa}$$

$$T_B = 1033 \text{ K}$$

B → C isochore

$$P_C = P_B$$

$$T_C = \gamma T_B$$

$$\Rightarrow T_C = 2T_B = 2066 \text{ K}$$

C → D adiabatic

$$V_D = V_A$$

$$P_D V_A^\gamma = P_C V_C^\gamma$$

$$P_D = P_C \left( \frac{V_C}{V_A} \right)^\gamma = P_C \left( \frac{\gamma}{r} \right)^\gamma = 2.64 \cdot 10^5 \text{ Pa}$$

$$\Rightarrow T_D = \frac{P_D V_A}{nR} = 790 \text{ K}$$

Rendimiento

$$\eta = 1 - \frac{1}{\gamma} \frac{T_D - T_A}{T_C - T_B} = 0.66 \Rightarrow 66\%$$

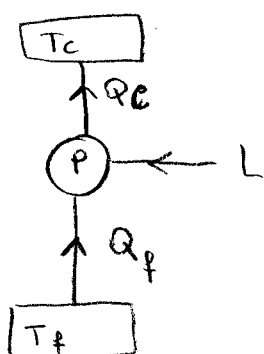
$$P_{\text{tot}} = 4 \times \frac{2000}{60} \times \Delta L \quad \frac{1}{2}$$

$$\Delta L = n C_p (T_C - T_B) + n C_v (T_A - T_D) = 845 \text{ J}$$

$$\Rightarrow P_{\text{tot}} = 56.3 \text{ kW.}$$

## Esercizio sulle pompe di calore

1



$$T_f < T_c$$

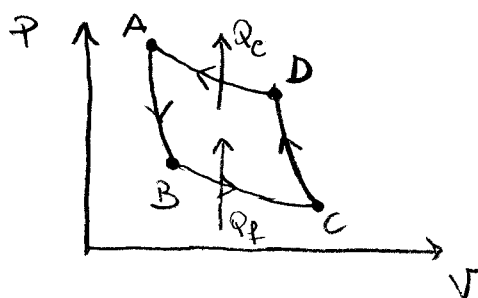
Pompa di calore: preleva calore da un ambiente più freddo a uno più caldo.

Può essere usata per riscaldare o per raffreddare.

$$\text{efficienza} = \frac{\text{quello che si ottiene}}{\text{lavoro fatto}} = \eta$$

$$\Rightarrow \eta = \frac{Q_c}{L} \quad \text{per riscaldare}$$

Supporremo adesso di avere una macchina di Carnot all'incontrario



$$Q_{BC} = Q_f = L_{BC} = n R T_f \ln \frac{V_C}{V_B} = - n R T_f \frac{V_B}{V_C}$$

$$Q_{DA} = Q_c = L_{DA} = n R T_c \ln \frac{V_A}{V_D}$$

$$\begin{matrix} T_A & V_A^{\gamma-1} & = & T_B & V_B^{\gamma-1} \\ T & \dots \gamma-1 & & T & \dots \gamma-1 \end{matrix} \Rightarrow \frac{V_A}{V_D} = \frac{V_B}{V_C}$$

$$\eta = \frac{L_{DA}}{L_{BE} + L_{DA}} = \frac{T_c}{T_c - T_f} = \frac{1}{1 - \frac{T_f}{T_c}}.$$

Per vedere dove

$$\eta = \frac{Q_f}{L} \Rightarrow \text{per Carnot}$$

$$\eta = \frac{T_c}{T_c - T_f}$$